[ENG29]

# UNIVERSITY OF BOLTON SCHOOL OF ENGINEERING

# MSC SYSTEMS ENGINEERING AND ENGINEERING MANAGEMENT

# **SEMESTER 2 EXAMINATION 2023/2024**

# **ADVANCED CONTROL TECHNOLOGY**

**MODULE NUMBER: EEM7015** 

Date: Tuesday 14<sup>th</sup> May 2024 Time: 10:00 – 12:00

INSTRUCTIONS TO CANDIDATES: There are FIVE questions.

Answer any FOUR questions.

All questions carry equal marks.

Marks for parts of questions are shown in

brackets.

This examination paper carries a total of

100 marks.

Formulae sheet is attached at the end of the

paper.

All working must be shown. A numerical solution to a question obtained by programming an electronic calculator will

not be accepted.

#### **Question 1**

A block diagram for a furnace temperature control system is shown in Figure Q1 below:

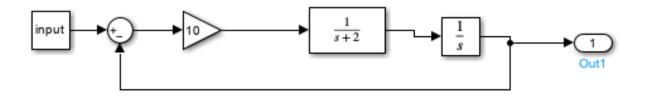


Figure Q1

- a) Determine the system damping ratio, natural frequency, damped frequency, and steady state gain.[7 marks]
- b) Determine the time domain response of the system, y(t), to a unit impulse input, r(t). [6 marks]
- c) For a unit step input, determine the system rise time, peak time, maximum percentage overshoot, and settling time for a 2% tolerance.

[6 marks]

d) If the input of  $r(t) = 100, t \ge 0$  and  $r(t) = 0, t \le 0$ 

is applied, analyse the system steady state error.

[6 marks]

Total marks [ 25 marks]

#### Question 2

A filter is shown in Figure Q2.

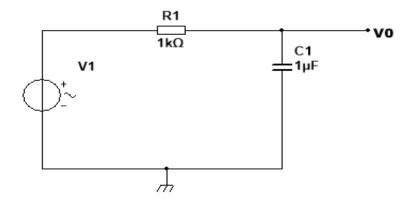


Figure Q2

Where Vo is the output and V1 is the input.

a) Determine the transfer function in the Laplace domain.

[8 marks]

b) Sketch the pole zero map, indicating the poles and zeros.

[5 marks]

c) Derive the equation that predicts the output voltage in the time domain.

[7 marks]

d) Is the filter stable and what type of filter is it.

[5 marks]

Total marks [ 25 marks]

#### **Question 3**

A dynamic system is shown in the block diagram below, Figure Q3(a),

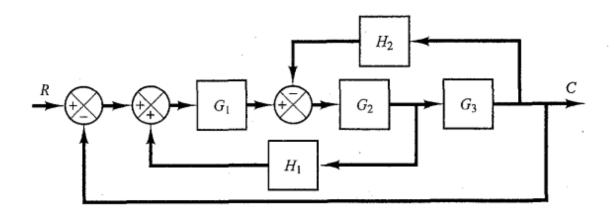


Figure Q3(a)

- a) Show how the block diagram in figure Q3(a) could be reduced to describe the output over the input C/R. [6 marks]
- b) From the Matlab graph shown in Q3(b), estimate the gain and phase margins [6 marks]
- c) Sketch the magnitude and phase for the following functions.

(i) 
$$G1(s) = \frac{10}{0.2s+1}$$
 [2 marks]

(ii) G2(s) = 
$$\frac{2500}{S^2 + 80S + 2500}$$
 [2 marks]

(iii) 
$$G3(s) = \frac{8}{s}$$
 [2 marks]

d) Sketch the final result of G1(s)\*G2(s)\*G(s) and estimate the phase and gain margin. [7 marks]

Question 3 continues over the page...

#### Question 3 continued...

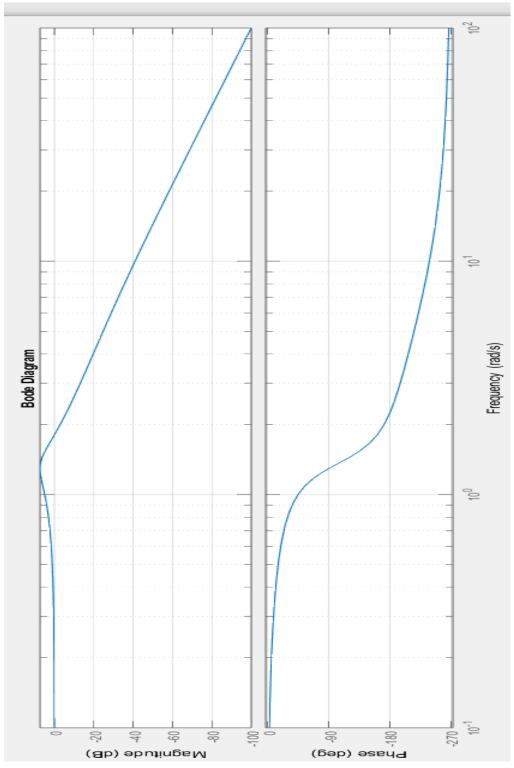


Figure 3(b)

Total marks [ 25 marks]

#### **Question 4**

A suspension system for mountain bicycle is shown Figure Q4. If the input is force F(t) and the output displacement Y(t);

- (a) Develop the system differential equation and transfer function. [15 marks]
- (b) Determine the state space equations and Matrices A,B,C and D [10 marks]

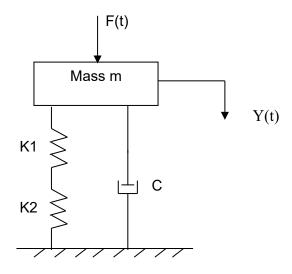
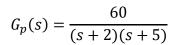


Figure Q4

Total marks [ 25 marks]

#### **Question 5**

A PID controller is used to control an automation processing plant as shown in Figure Q5. The open loop transfer function of the plant is given by.



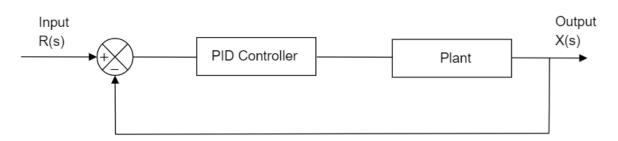


Figure Q5: Control system of the processing plant.

(a) Evaluate the performances of closed loop plant system (natural frequency, damping ratio, Percentage Overshoot, peak time, rise time, settling time and steady-state error) to assess its performance without the PID controller.

[10 marks]

- (b) Design a PID controller to determine the parameter  $K_p$ ,  $K_i$  and  $K_d$ , and clearly identify the design procedure if the system responses for a unit step input are required to be:
  - The maximum overshoot is less than 8%.
  - The settling time is 40% less than without the PID controller.
  - The steady-state error is 0.

[15 marks]

Total marks [ 25 marks]

#### **END OF QUESTIONS**

Formulae sheets follow over the page

#### Formula sheet

#### Blocks with feedback loop

$$G(s) = \frac{Go(s)}{1 + Go(s)H(s)}$$
 (for a negative feedback)

$$G(s) = \frac{Go(s)}{1 - Go(s)H(s)}$$
 (for a positive feedback)

# Steady-State Errors

$$e_{ss} = \lim_{s \to 0} \left[ s \frac{1}{1 + G_o(s)} \theta_i(s) \right]$$
 (for the closed-loop system with a unity feedback)

$$e_{H} = \lim_{s \to 0} \left[ s \frac{1}{1 + \frac{G_0(s)}{1 + G_0(s)[H(s) - 1]}} \theta_i(s) \right] \text{ (if the feedback H(s) } \neq 1 \text{)}$$

$$e_{zz} = \frac{1}{1 + \lim_{z \to 1} G_0(z)}$$
 (if a digital system subjects to a unit step input)

# **Laplace Transforms**

A unit impulse function

A unit step function

A unit ramp function  $\frac{1}{s^2}$ 

### First order Systems

$$G(s) = \frac{\theta_o}{\theta_s} = \frac{G_{ss}(s)}{\tau s + 1}$$

$$\tau \left(\frac{d\theta_o}{dt}\right) + \theta_o = G_{ss}\theta_i$$

$$\theta_O = G_{\omega} (1 - e^{-\epsilon/\tau})$$
 (for a unit step input)

$$\theta_0 = AG_{ss}(1 - e^{-\epsilon/\tau})$$
 (for a step input with size A)

$$\theta_o(t) = G_{ss}(\frac{1}{t})e^{-(t/\tau)}$$
 (for an impulse input)

#### Second-order systems

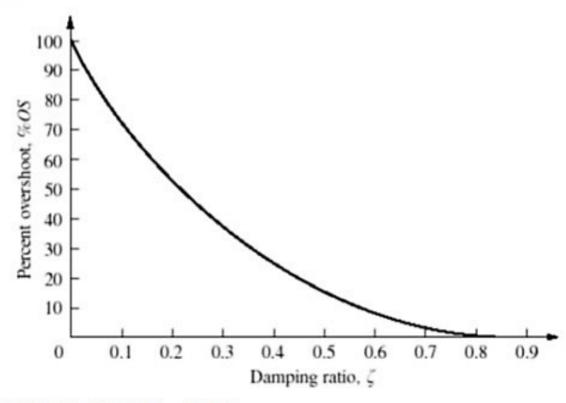
$$\frac{d^{2}\theta_{o}}{dt^{2}} + 2\zeta\omega_{n}\frac{d\theta_{o}}{dt} + \omega_{n}^{2}\theta_{o} = b_{o}\omega_{n}^{2}\theta_{i}$$

$$G(s) = \frac{\theta_o(s)}{\theta_i(s)} = \frac{b_o \omega_s^2}{s^2 + 2\zeta \omega_n s + \omega_s^2}$$

$$\omega dt = 1/2\pi$$
  $\omega dt = \pi$ 

P.O. = 
$$\exp(\frac{-\zeta \pi}{\sqrt{(1-\zeta^2)}}) \times 100\%$$

$$t_s = \frac{4}{\zeta \omega_n}$$
  $\omega_d = \omega_n \sqrt{(1-\zeta^2)}$ 



Controllability: R = [B AB A2B ..... A(n-1) B]

# Laplace transform and Z transform table

| Laplace Domain          | Time Domain              | Z Domain  |
|-------------------------|--------------------------|---|
| 1                       | $\delta(t)$ unit impulse | 1   |
| 1                       | u(t) unit step           |   |
| S                       |                          | z-1   |
| 1                       | t                        | Tz  |
| $s^2$                   | t                        | $(z-1)^2$                                       |
| 1                       | $e^{-at}$                | Z   |
| s + a                   |                          | $z - e^{-aT}$                                   |
| $\frac{1}{s(s+a)}$      | $\frac{1}{a}(1-e^{-at})$ | $\frac{z(1-e^{-aT})}{a(z-1)(z-e^{-aT})}$        |
| b-a                     | $e^{-at} - e^{-bt}$      | $\frac{a(z-1)(z-e^{-t})}{z(e^{-aT}-e^{-bT})}$   |
| $\overline{(s+a)(s+b)}$ | C                        | $\frac{2(c-c)}{a(z-e^{aT})(z-e^{-bT})}$         |
| b                       | $e^{-at}\sin(bt)$        | $ze^{-aT}\sin(bT)$                              |
| $(s+a)^2+b^2$           |                          | $\overline{z^2 - 2ze^{-aT}\cos(bT) + e^{-2aT}}$ |
| s+a                     | $e^{-at}\cos(bt)$        | $z^2 - ze^{-aT}\cos(bT)$                        |
| $(s+a)^2+b^2$           |                          | $z^2 - 2ze^{-aT}\cos(bT) + e^{-2aT}$            |

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Laplace Transforms of common functions

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|-------------------------|-----------------------------|---|
| Functions               |                             |   |
| Unit pulse (Dirac delta | $\delta(t)$                 | F(s)=1  |
| distribution)           |                             |   |
| Unit step function      | 1(t)                        | $F(s) = \frac{1}{s}$                                  |
| Ramp function           | f(t) = at                   | $F(s) = \frac{1}{s^2}$                                |
| Sine function           | $f(t) = \sin at$            | $F(s) = \frac{a}{s^2 + a^2}$                          |
| Cosine function         | $f(t) = \cos at$            | $F(s) = \frac{s}{s^2 + a^2}$                          |
| Exponential function    | $f(t)=e^{at}$               | $F(s) = \frac{s}{s^2 + a^2}$ $F(s) = \frac{1}{s - a}$ |
| Operations              |                             |   |
| Differentiation         | L(f'(t))                    | sF(s)-f(0)  |
| Integration             | $L\left(\int f(t)dt\right)$ | $\frac{1}{s}F(s)$                                     |
| Time shift              | Lf(t-a)                     | $e^{-as}F(s)$   |