UNIVERSITY OF BOLTON

OFF CAMPUS DIVISION

STUDY WORLD LANKA CAMPUS

BENG(HONS) MECHANICAL ENGINEERING

SEMESTER ONE EXAMINATION 2018/2019

ADVANCED THERMOFLUIDS & CONTROL SYSTEM

MODULE NO: AME6005

Date: 30TH September 2018

Time: 09:00am – 11:00am

INSTRUCTIONS TO CANDIDATES:

There are 6 questions.

Answer 4 questions.

All questions carry equal marks. Attempt Three questions from PART A and one questions from PART B

Marks for parts of questions are shown in brackets.

Thermodynamic properties of fluids tables are provided

CANDIDATES REQUIRE :

PART A

Q1. a) Water at 25°C flows steadily through the box shown in Figure Q1a, entering station-A at 2 m/s. Both entering and leaving stations are circular pipes with diameters 9cm and 4cm respectively. Calculate the horizontal and vertical forces required to hold the system against the momentum.



b) The pressure drop per unit length Δp/L in a porous, rotating duct depends upon average velocity U, density ρ, viscosity μ, duct height h, wall injection velocity U_i, and rotation rate ω. Considering ρ, U and h as repeating variables defines in Buckingham pi theorem, express this relationship in dimensionless form.

> (13 marks) Total 25 marks

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- Q2. a) Briefly explain following terms related to internal duct (pipe) flow.
 - i.Laminar and Turbulent flow
 - ii. Velocity boundary layer
 - iii. Entrance region
 - iv. Fully developed flow

(8 marks)

b) The system in Figure Q2b consists of 800 m of 4 cm cast-iron pipe, two 45° and six 90° flanged long-radius elbows, one fully open flanged globe valve, one gate valve, one angle valve and a sharp exit into a reservoir. If the elevation at point A is 100 m, what gage pressure is required at point A to deliver 0.006 m³/s of water at 25°C into the reservoir?



(17 marks) Total 25 marks

Q3. (a) A piston–cylinder device contains 0.8 kg of atmospheric air initially at 100 kPa and 27°C. The atmospheric air is now compressed slowly in a Polytropic manner process during which $PV^{1.3}$ = constant until the volume is reduced by one-third. Determine the work done and the heat transfer for this process.

(13 marks)

- (b) Steam flows steadily through an adiabatic turbine. The inlet conditions of the steam are 10 MPa, 450°C, and 100 m/s, and the exit conditions are 10 kPa, 95 percent quality, and 30 m/s. The mass flow rate of the steam is 16 kg/s. Determine
 - i.The change in kinetic energy
 - ii.The power output
 - iii.The turbine inlet and outlet areas



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Q4. The gas-turbine portion of a combined gas-steam power plant has a pressure ratio of 15. Air enters the compressor at 300 K at a rate of 14 kg/s and is heated to 1500 K in the combustion chamber. The combustion gases leaving the gas turbine are used to heat the steam to 400°C at 10 Mpa in a heat exchanger. The combustion gases leave the heat exchanger at 420 K. The steam leaving the turbine is condensed at 15 kPa. Assuming all the compression and expansion processes to be isentropic, For air, assume constant specific heats at room temperature. Determine,

i. Air compressor and gas turbine outlet air temperatures

- ii. Gas power cycle net power output
- iii. The mass flow rate of the steam
- iv. The net power output of steam power cycle
- v. the thermal efficiency of the combined cycle.



Figure Q4

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<u>PART B</u>

Q5. (a) Describe the importance of stability in control systems.

(4 marks)

(b) For an automated control system open loop transfer function is given as follows. Considering unity feedback control,

$$G(s) = \left(K_P + \frac{K_I}{s}\right) \frac{1}{s(s+2)}$$

- i. Find the characteristic equation of the system
- ii. Referring RH criteria find the condition of stability related to KP and KI.

(12 marks)

I Draw the bode plot for below transfer function

$$G(s)H(s) = \frac{14(s+9)^2}{(s+2)(s+5)(s+7)(s+13)}$$

(9 marks)

Total 25 marks

Q6. (a) Develop a state model for below transfer function following diagonal method used in state space analysis.

$$TF = \frac{1}{(s+2)(s+5)(s+7)(s+13)}$$

(15 marks)

- (b) i. Explain the physical meaning of PID controller referring sketches graphs.
 - ii. Write down three advantages of PID controller

(10 marks)

Total 25 marks

END OF QUESTIONS

FORMULA SHEET

P = F/A, $\rho = m/v$, $m = \rho AV$, $P = P_g + P_{atm}$, $P = \rho gh$ Q- W = $\triangle U + \triangle PE + \triangle KE$, W = $\int PdV$, P Vⁿ = C $\mathbf{W} = \frac{\mathbf{P}_1 \, \mathbf{V}_1 - \mathbf{P}_2 \, \mathbf{V}_2}{\mathbf{n} - 1}$ $W = P(v_2 - v_1)$ $W = PV \ln\left(\frac{V_2}{V_1}\right)$ $Q = C_d A \sqrt{2gh}$ $V_1 = C \sqrt{2g h_2 \left(\frac{\rho g_m}{\rho g} - 1\right)}, \quad \sum F = \frac{\Delta M}{\Delta t} = \Delta M, \quad F = \rho QV, \quad T = -(\partial p/\partial x) r/2$ $Re = V D \rho/\mu, \quad \Delta p = (32\mu VL)/D^2, \quad Q = du + dw, \quad du = Cv dT, \quad dw = pdv$ Re = V D ρ/μ , $\Delta p - (32\mu + L_f) =$ pv = mRT, $h = h_f + xhf_g$, $s = s_f + xsf_g$, v = x Vg, $Q - w = \sum mh$, $F = \frac{2\pi L\mu}{L_n \left(\frac{R_2}{R_2}\right)}$ $F_D = \frac{1}{2}CD \ \rho u^2 s$, $F_L = \frac{1}{2}C_L \rho u^2 s$, $S_p = \frac{d}{ds}(P + \rho gZ)$, $Q = \frac{\pi D^4 \Delta p}{128\mu L}$ $h_{f} = \frac{4fLv^{2}}{d2g}$, $h_{m} = \frac{Kv^{2}}{2g}$, $h_{m} = \frac{k(V_{1} - V_{2})^{2}}{2g}$ $f = \frac{16}{2g}$ $h_f = \frac{64}{R} \bigg($

$$\eta = \left(1 - \frac{T_L}{T_H}\right), \quad \eta = (h_1 - h_2)/(h_1 - h_f 2), \qquad S_{gen} = (S_2 - S_1) + \frac{Q}{T}$$

 $F = \tau \pi DL$

Blocks with feedback loop

$$G(s) = \frac{Go(s)}{1 + Go(s)H(s)}$$
 (for a negative feedback)

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 $G(s) = \frac{Go(s)}{1 - Go(s)H(s)}$ (for a positive feedback)

Steady-State Errors

 $e_{ss} = \lim_{s \to 0} [s(1 - G_O(s))\theta_i(s)]$ (for an open-loop system)

 $e_{ss} = \lim_{s \to 0} [s \frac{1}{1 + G_o(s)} \theta_i(s)]$ (for the closed-loop system with a unity feedback)



Material	Condition	ft	mm	Uncertainty, %	
Steel	Sheet metal, new	0.00016	0.05	±60	
	Stainless, new	0.000007	0.002	± 50	
	Commercial, new	0.00015	0.046	±30	
	Riveted	0.01	3.0	± 70	
	Rusted	0.007	2.0	±50	
Iron	Cast, new	0.00085	0.26	±50	
	Wrought, new	0.00015	0.046	± 20	
	Galvanized, new	0.0005	0.15	± 40	
	Asphalted cast	0.0004	0.12	± 50	
Brass	Drawn, new	0.000007	0.002	± 50	
Plastic	Drawn tubing	0.000005	0.0015	± 60	
Glass	_	Smooth	Smooth		
Concrete	Smoothed	0.00013	0.04	± 60	
	Rough	0.007	2.0	±50	
Rubber	Smoothed	0.000033	0.01	± 60	
Wood	Stave	0.0016	0.5	± 40	

Recommended Roughness Values for Commercial Ducts

Nominal diameter, in										
Screwed				Flanged						
$\frac{1}{2}$	1	2	4	1	2	4	8	20		
14	8.2	6.9	5.7	13	8.5	6.0	5.8	5.5		
0.30	0.24	0.16	0.11	0.80	0.35	0.16	0.07	0.03		
5.1	2.9	2.1	2.0	2.0	2.0	2.0	2.0	2.0		
9.0	4.7	2.0	1.0	4.5	2.4	2.0	2.0	2.0		
0.39	0.32	0.30	0.29							
				0.21	0.20	0.19	0.16	0.14		
2.0	1.5	0.95	0.64	0.50	0.39	0.30	0.26	0.21		
1.0	0.72	0.41	0.23	0.40	0.30	0.19	0.15	0.10		
2.0	1.5	0.95	0.64	0.41	0.35	0.30	0.25	0.20		
				0.40	0.30	0.21	0.15	0.10		
0.90	0.90	0.90	0.90	0.24	0.19	0.14	0.10	0.07		
2.4	1.8	1.4	1.1	1.0	0.80	0.64	0.58	0.41		
	$\frac{1}{2}$ 14 0.30 5.1 9.0 0.39 2.0 1.0 2.0 0.90 2.4	1 8.2 0.30 0.24 5.1 2.9 9.0 4.7 0.39 0.32 2.0 1.5 1.0 0.72 2.0 1.5 0.90 0.90 2.4 1.8	Screwed 1 2 14 8.2 6.9 0.30 0.24 0.16 5.1 2.9 2.1 9.0 4.7 2.0 0.39 0.32 0.30 2.0 1.5 0.95 1.0 0.72 0.41 2.0 1.5 0.95 1.4 1.8 1.4	Image: Screwed Image: Screwed Image: I	Image: Screwed Image: Screwed Image: I	Image: Screwed Image: Screwed I <thi< th=""> I<!--</td--><td>Nominal diameter, in Screwed Flanger 1/2 1 2 4 1 2 4 14 8.2 6.9 5.7 13 8.5 6.0 0.30 0.24 0.16 0.11 0.80 0.35 0.16 5.1 2.9 2.1 2.0 2.0 2.0 2.0 2.0 9.0 4.7 2.0 1.0 4.5 2.4 2.0 0.39 0.32 0.30 0.29 0.21 0.20 0.19 2.0 1.5 0.95 0.64 0.50 0.39 0.30 1.0 0.72 0.41 0.23 0.40 0.30 0.19 2.0 1.5 0.95 0.64 0.41 0.35 0.30 1.0 0.72 0.41 0.23 0.40 0.30 0.21 0.90 0.90 0.90 0.90 0.24 0.19 0.14<td>Nominal diameter, in Screwed Flanged 1/2 1 2 4 1 2 4 8 14 8.2 6.9 5.7 13 8.5 6.0 5.8 0.30 0.24 0.16 0.11 0.80 0.35 0.16 0.07 5.1 2.9 2.1 2.0</td></td></thi<>	Nominal diameter, in Screwed Flanger 1/2 1 2 4 1 2 4 14 8.2 6.9 5.7 13 8.5 6.0 0.30 0.24 0.16 0.11 0.80 0.35 0.16 5.1 2.9 2.1 2.0 2.0 2.0 2.0 2.0 9.0 4.7 2.0 1.0 4.5 2.4 2.0 0.39 0.32 0.30 0.29 0.21 0.20 0.19 2.0 1.5 0.95 0.64 0.50 0.39 0.30 1.0 0.72 0.41 0.23 0.40 0.30 0.19 2.0 1.5 0.95 0.64 0.41 0.35 0.30 1.0 0.72 0.41 0.23 0.40 0.30 0.21 0.90 0.90 0.90 0.90 0.24 0.19 0.14 <td>Nominal diameter, in Screwed Flanged 1/2 1 2 4 1 2 4 8 14 8.2 6.9 5.7 13 8.5 6.0 5.8 0.30 0.24 0.16 0.11 0.80 0.35 0.16 0.07 5.1 2.9 2.1 2.0</td>	Nominal diameter, in Screwed Flanged 1/2 1 2 4 1 2 4 8 14 8.2 6.9 5.7 13 8.5 6.0 5.8 0.30 0.24 0.16 0.11 0.80 0.35 0.16 0.07 5.1 2.9 2.1 2.0		

Resistance Coefficients $K = h_m/[V^2/(2g)]$ for Open Valves, Elbows, and Tees